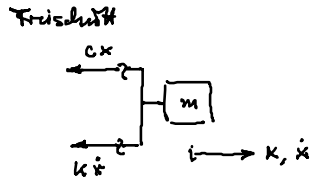
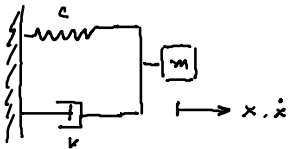


- Nichterhaltung gedämpfte Schwingungen
- Angeführte Schwingung
- Resonanz mit/ohne Dämpfung
- Phasenverschiebung

Freiübertragung



Gleichung
 $m \ddot{x} = -kx - c\dot{x}$

Normalform
 $\ddot{x} + \underbrace{\frac{c}{m}}_{2\delta} \dot{x} + \underbrace{\frac{k}{m}}_{\omega^2} x = 0$

Lösung: $x(t) = A \exp(\lambda t)$
 $\dot{x}(t) = \lambda x(t)$
 $\ddot{x}(t) = \lambda^2 x(t)$ } $\rightarrow x(t) [\lambda^2 + 2\delta\lambda + \omega^2] = 0$

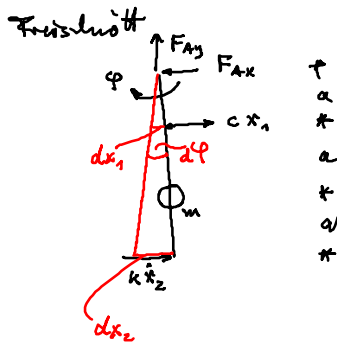
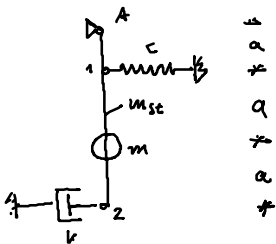
$\Rightarrow \lambda^2 + 2\delta\lambda + \omega^2 = 0$ char. Gleichung

$\lambda_{1,2} = -\delta \pm \sqrt{\delta^2 - \omega^2} = -\delta \pm \omega \sqrt{\left(\frac{\delta}{\omega}\right)^2 - 1}$

D^2 (Lehrscher Dämpfungsmaß)

- 1) $D > 1$ starke Dämpfung
- 2) $D = 1$ aperiodischen Grenzfall
- 3) $0 < D < 1$ Schwingfall

Beispiel



Gleichungen
 Drehmomente um A
 $-\theta^{(A)} \ddot{\varphi} = c x_1 + k x_2$
 $\theta^{(A)} = \frac{m \delta a}{3} (3a)^2 + m (2a)^2$

kinematische Bedingungen

$3a d\varphi = dx_2 \Rightarrow \ddot{x}_2 = 3a \ddot{\varphi}$
 $a d\varphi = dx_1 \Rightarrow \dot{x}_1 = a \dot{\varphi}$

$\theta^{(A)} \ddot{\varphi} + k(3a)^2 \varphi + c a^2 \varphi = 0$
 $\ddot{\varphi} + \underbrace{\frac{k 9 a^2}{\theta^{(A)}}}_{2\delta} \dot{\varphi} + \underbrace{\frac{c a^2}{\theta^{(A)}}}_{\omega^2} \varphi = 0$

A.B: $\varphi(0) = \varphi_0$

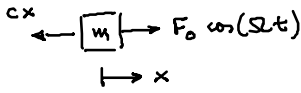
3.4.4 Angeführte Schwingungen

(a) ohne Dämpfung

$$F(t) = F_0 \cos(\Omega t)$$

$$\dot{\varphi}(0) = \omega$$

FS:



Gleichungen

$$m \ddot{x} = -c x + F_0 \cos(\Omega t)$$

$$\ddot{x} + \omega^2 x = \frac{c}{m} \frac{F_0}{c} \cos(\Omega t) = \frac{F_0}{m} \cos(\Omega t)$$

$$\Rightarrow \ddot{x} + \omega^2 x = \omega^2 x_0 \cos(\Omega t)$$

$$x(t) = \underbrace{x_{\text{hom}}(t)} + x_{\text{part}}(t) = C \cos(\omega t - \alpha)$$

Für die Partikulär-Lösung setzen wir an

$$x_{\text{part}}(t) = x_0 V \cos(\Omega t)$$

$$\dot{x}_{\text{part}} = -x_0 V \Omega \sin(\Omega t)$$

$$\ddot{x}_{\text{part}} = -x_0 V \Omega^2 \cos(\Omega t)$$

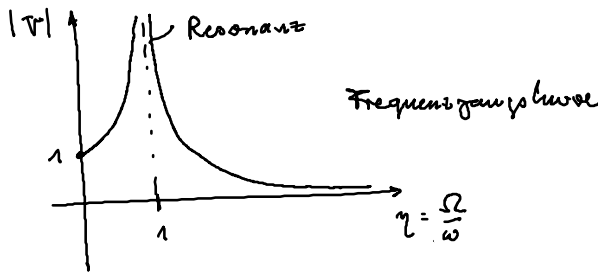
einsetzen in

$$\ddot{x}_{\text{part}} + \omega^2 x_{\text{part}} = \omega^2 x_0 \cos(\Omega t)$$

$$-x_0 V \Omega^2 \cos(\Omega t) + \omega^2 x_0 V \cos(\Omega t) = \omega^2 x_0 \cos(\Omega t)$$

$$\Rightarrow V = \frac{\omega^2}{\omega^2 - \Omega^2} = \frac{1}{1 - \left(\frac{\Omega}{\omega}\right)^2}$$

Abstimmung.
Frequenzverhältnis



$$\text{in } \ddot{x}_{\text{part}} + \omega^2 x_{\text{part}} = \omega^2 x_0 \cos(\Omega t)$$

$$2 x_0 V \Omega \cos(\Omega t) - x_0 V \Omega^2 \sin(\Omega t) + x_0 V \Omega^2 \sin(\Omega t) = \Omega^2 x_0 \cos(\Omega t)$$

$$\Rightarrow V = \frac{\Omega}{2} = \frac{\omega}{2}$$

Lösung für den Resonanzfall $\Omega = \omega$

$$x_{\text{part}} = x_0 \bar{V} t \sin(\Omega t) = x_0 \bar{V} t \sin(\omega t)$$

$$\dot{x}_{\text{part}} = x_0 \bar{V} \sin(\Omega t) + x_0 \bar{V} t \Omega \cos(\Omega t)$$

$$\ddot{x}_{\text{part}} = 2 x_0 \bar{V} \Omega \cos(\Omega t) - x_0 \bar{V} t \Omega^2 \sin(\Omega t)$$

$$\Rightarrow x_p = \frac{1}{2} x_0 t \sin(\Omega t)$$

AB's : $x(t=0) = 0$

$\dot{x}(t=0) = 0$

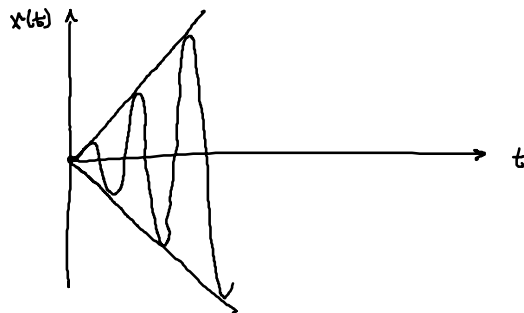
$$x(t) = C \cos(\omega t - \alpha) + \frac{x_0}{2} t \sin(\Omega t)$$

$$x(0) = C \cos(\alpha) \stackrel{!}{=} 0 \Rightarrow C = 0$$

$$\dot{x}(t) = -C \omega \sin(\omega t - \alpha) + \frac{x_0}{2} \sin(\Omega t) + \frac{x_0}{2} t \Omega \cos(\Omega t)$$

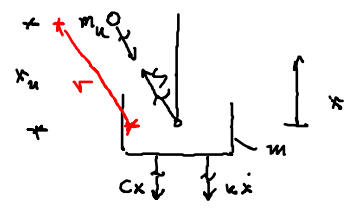
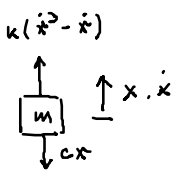
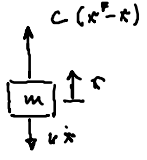
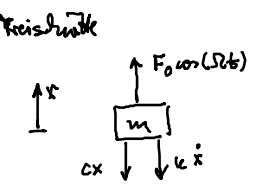
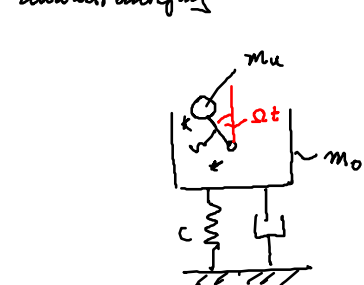
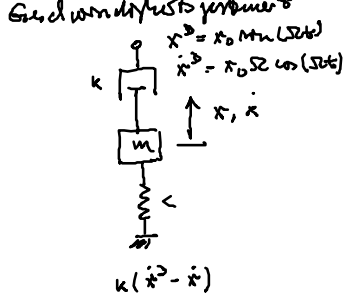
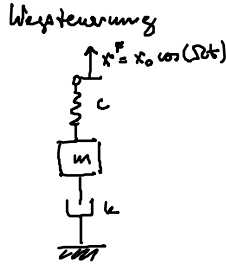
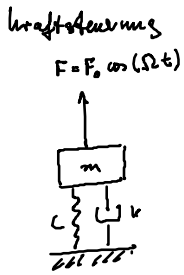
$$\dot{x}(0) = 0 + 0 + 0 = 0 \checkmark$$

$$\Rightarrow x(t) = \frac{x_0}{2} t \sin(\Omega t)$$



Anfachende Schwingungen mit geschwindigkeitsdep. Dämpfung

Vier Fälle



$$m \ddot{x} = -c x - k \dot{x} + F_0 \cos(\Omega t)$$

$$m \ddot{x} = -k \dot{x} + c(x^F - x)$$

$$m \ddot{x} = -c x + k(\dot{x} - \dot{x}^F)$$

$$m_0 \ddot{x} = -c x - k \dot{x} + S \cos(\Omega t)$$

$$\ddot{x} + 2\delta \dot{x} + \omega^2 x = \omega^2 x_0 \cos(\Omega t)$$

$\frac{k}{m}$ $\frac{c}{m}$ $\frac{F_0}{m}$

$$\ddot{x} + 2\delta \dot{x} + \omega^2 x = \frac{c}{m} x^F = \frac{c}{m} x_0 \cos(\Omega t)$$

$$\ddot{x} + 2\delta \dot{x} + \omega^2 x = 2D\eta \omega^2 x_0 \cos(\Omega t)$$

$\frac{k}{m}$ $\frac{c}{m}$ $\frac{S}{\omega}$

$$m_u \ddot{x}_u = -S \cos(\Omega t)$$

$$\ddot{x}_u = -\tau \cos(\Omega t) + \ddot{x}$$

$$\Rightarrow \ddot{x} + 2\delta \dot{x} + \omega^2 x = \omega^2 x_0 \cos(\Omega t)$$

$$(m_0 + m_u) \ddot{x} + k \dot{x} + c x = \tau \Omega^2 \cos \Omega t$$

Ergebnis: Alle 4 Fälle führen zu derselben DGL, nämlich

$$\frac{1}{\omega^2} \ddot{x} + \frac{2D}{\omega} \dot{x} + x = x_0 E \cos(\Omega t) \quad \text{mit } E = \begin{cases} = 1 & \text{für Kraft/Wege} \\ = 2D\eta & \text{für Geschw. st.} \\ = \eta^2 & \text{für Umwandl.} \end{cases}$$

Zur Lösung $x(t) = x_h(t) + x_p(t)$
 $\rightarrow 0$ für $t \rightarrow \infty$

$x_p(t) = x_0 V \cos(\Omega t - \varphi)$ einsetzen und V und φ - Bestimmungsgl. umstellen

$$\Rightarrow \tan \varphi = \frac{2D\eta}{1 - \eta^2}, \quad V = \frac{E}{\sqrt{(1 - \eta^2)^2 + 4D^2\eta^2}}$$